



**SIMPACK User Meeting
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Towards railway wheel performance prediction by simulation

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Outline

- **Background**
- **Wheel-rail interface performance**
- **Emerging prediction methods**
 - Wear prediction
 - Rolling contact fatigue assessment
- **SIMPACK implementation**
- **Applications**
 - Hollow wear
 - Rolling contact fatigue
 - Accumulated fatigue damage
- **Wheel-rail contact issues**
- **Conclusions**



Market requirements

- **Increased performance requirements on wheels**
 - Higher axle loads
 - Higher braking and traction forces
 - Higher dynamic loads due higher speeds
 - Smaller wheels
- **Increased focus on maintenance costs**
 - Wheel deterioration is in the top 10 of maintenance cost drivers
- **Track access charges being differentiated towards track damage dependency**
- **Increased interoperability between different track systems with different rail standards**

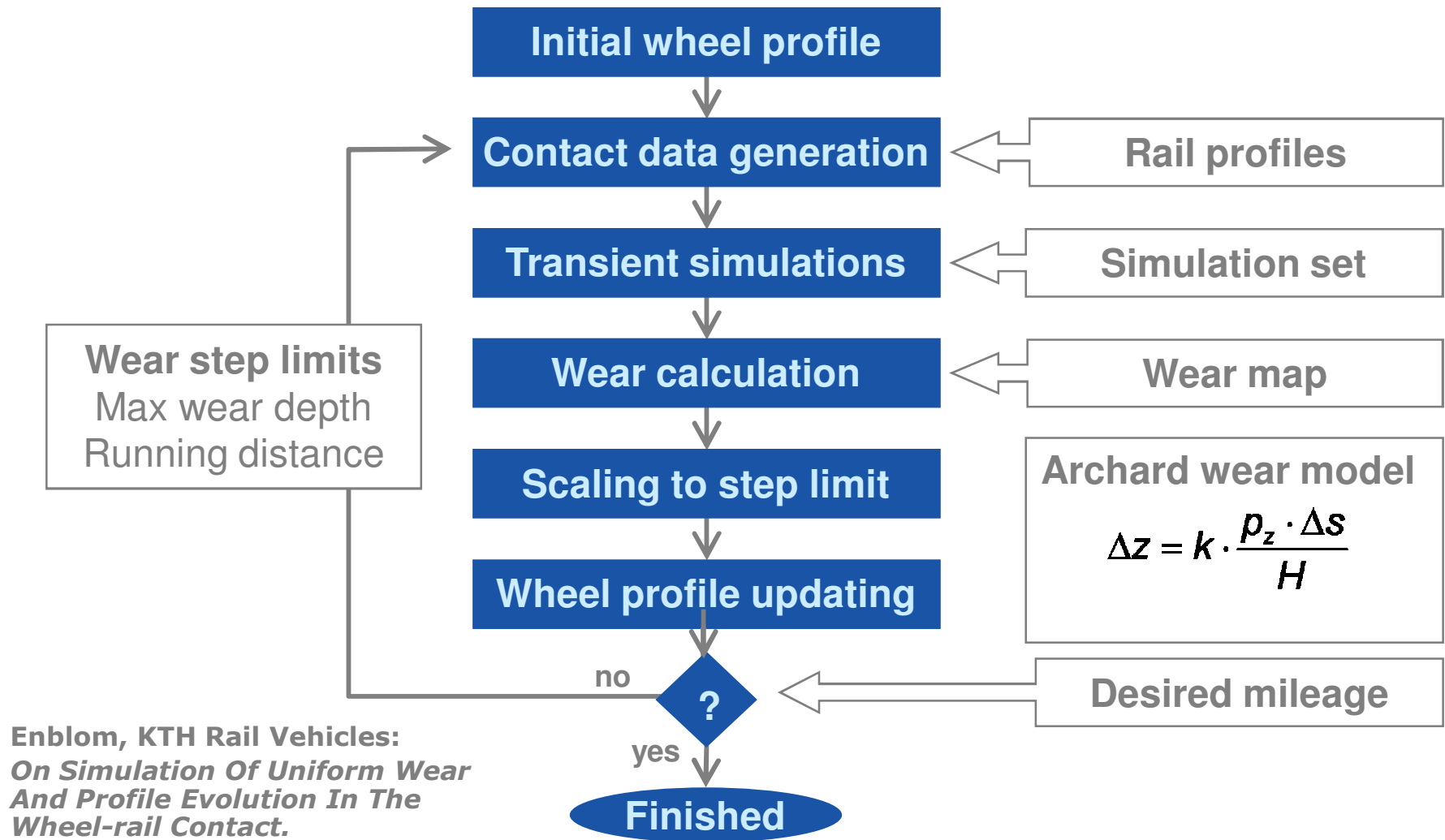
Modes of deterioration

- **Rolling contact fatigue (RCF)**
 - Shelling, spalling, head checking, fracture
- **Adhesive or abrasive wear**
 - Material loss, altered profile geometry
- **Plastic deformation**
 - Material relocation, ratchetting
- **Phase transformation**
 - Altered material properties, martensite
- **Mode interactions**
 - Out-of-roundness, corrugation

Emerging technologies

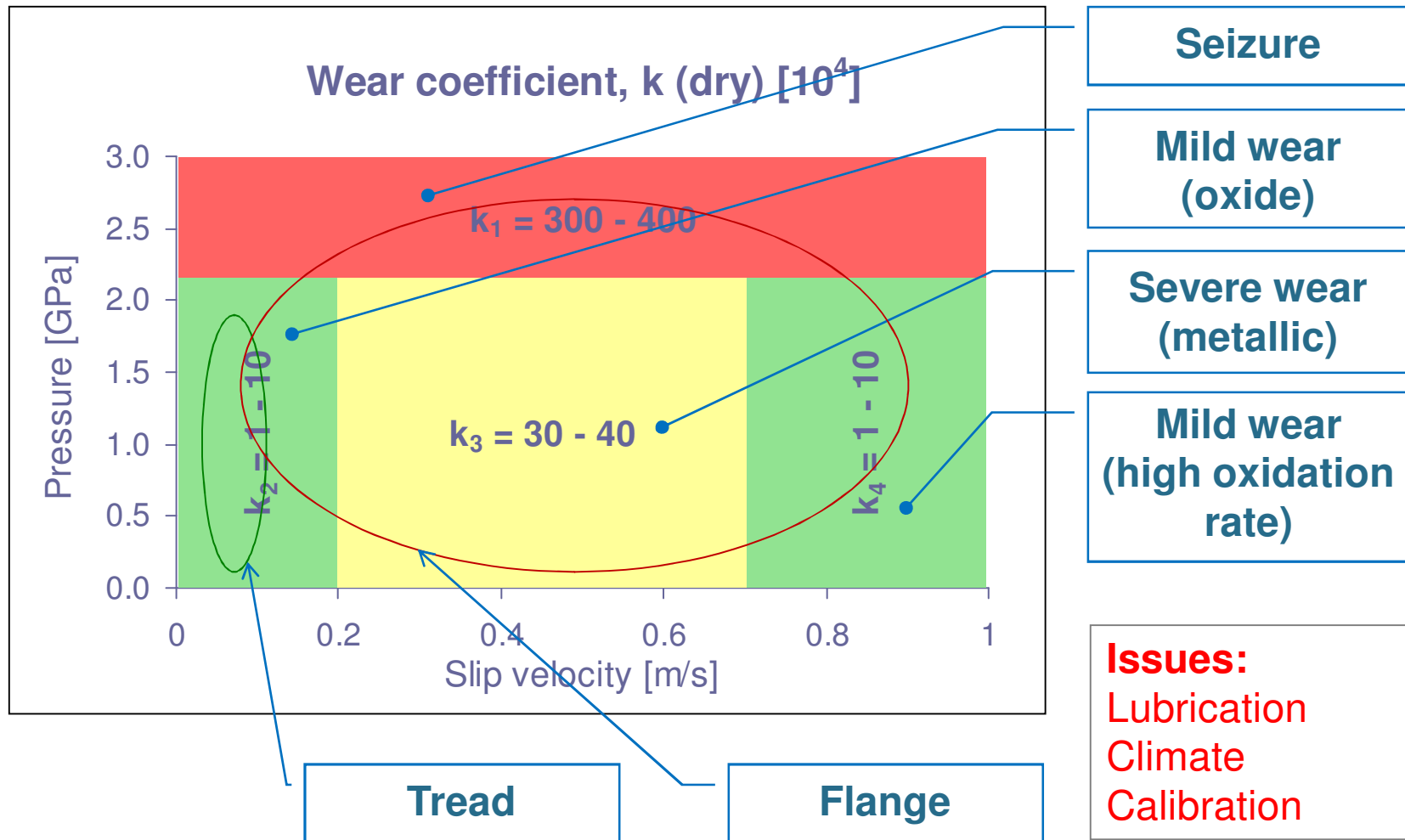
- **Wheel profile wear prediction procedure based on Archard's wear model**
(Royal Institute of Technology, Sweden)
- **Empirical RCF risk assessment method including influence of wear**
(Delta Rail for RSSB, UK)
- **Engineering model for RCF risk assessment based on shake down properties**
(Chamers Technical University, Sweden)

Wheel wear simulation procedure



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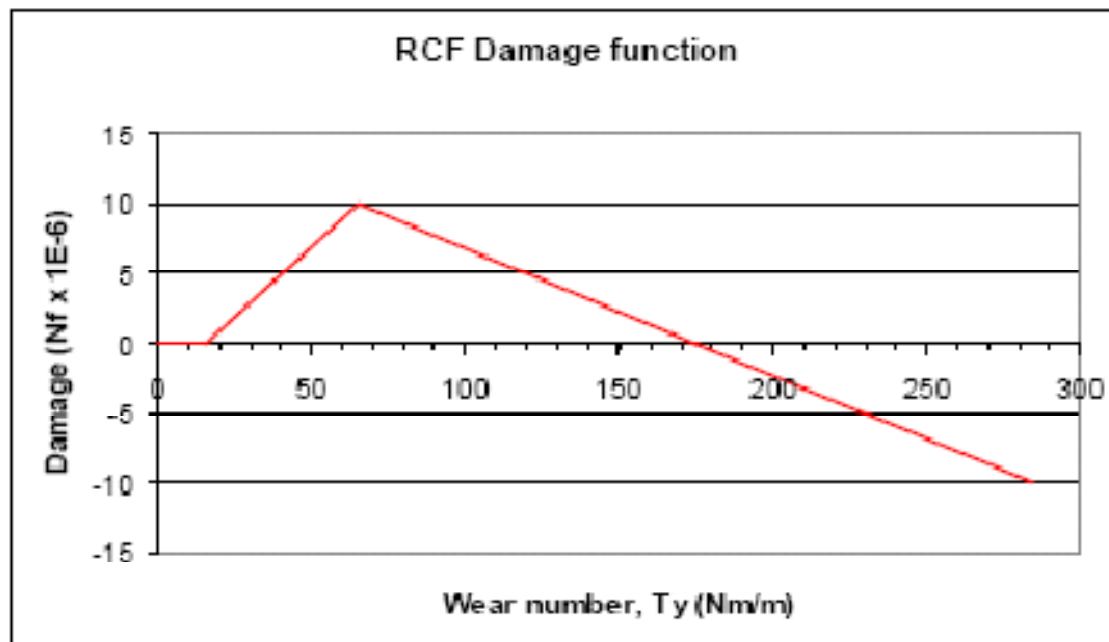
Map of wear coefficients



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Empirical RCF damage model

Combined fatigue/wear damage related to energy dissipation

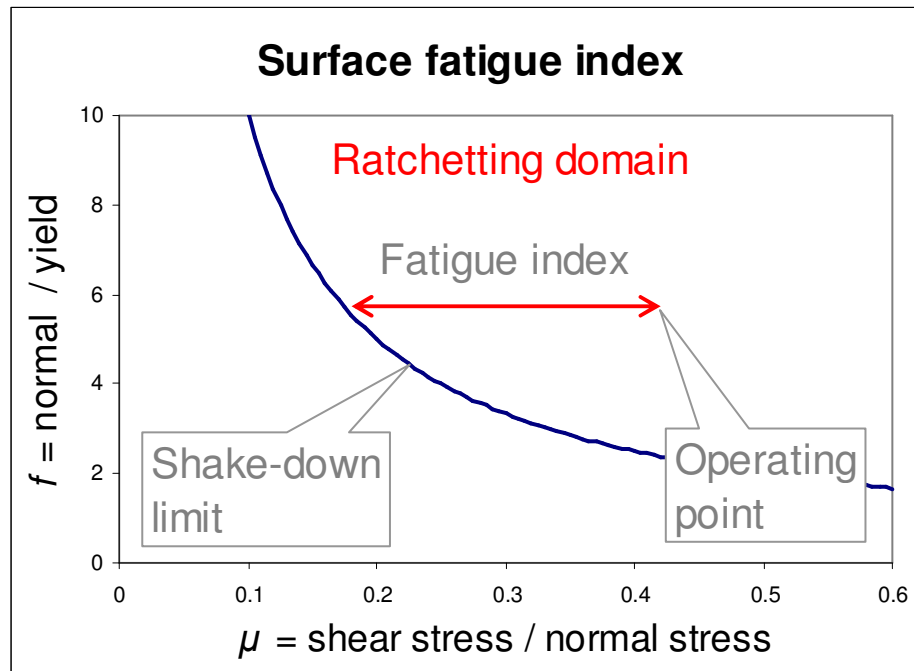


Issues:
Fatigue life
Calibration
Contact pressure

Burstow, Delta Rail (for RSSB):

*Whole Life Rail Model Application and Development for RSSB – Continued
Development of an RCF Damage Parameter*

Engineering criterion for surface fatigue



Risk criterion:

$$FI_{\text{surf}} = \mu - p_0/k < 0$$

μ = traction coefficient

p_0 = contact pressure

k = yield strength in shear

Issues:

Fatigue life

Crack depth

Creep

Ekberg, Kabo, Chalmers Applied Mechanics :

An Engineering Criterion for Prediction of Surface Initiated Rolling Contact Fatigue.

SIMPACK implementation

- **SIMPACK development (public parts)**
 - Internal profile and wear data handling
 - User routine interface for wear model implementation
 - Krause & Poll wear model
 - Wear accumulation and profile updating
 - Extension of the parameter variation facility to handle the wear simulation set and loop control

SIMPACK implementation

- **BT development (proprietary parts)**
 - Implementation of wear model as proprietary user functions
 - Contact patch discretisation and local contact condition calculation
 - Archard's wear model
 - Automatic wear step control
 - Running distance calculation
 - Implementation of RCF criteria as proprietary user functions
 - Shake down model
 - RSSB damage function
 - Software procedure testing
 - Quantitative validation with reference operations

Representation in SIMPACK Virtual Test Lab

- **Simulation set in the innermost loop**
 - Controlled by ParVar steering files
 - Selecting predefined database tracks
 - Selecting track irregularity files
 - Further parameters to be varied in parallel, for instance speed, weighting factors, simulation time, number of output points, ...
 - Varying of rail profiles
 - Variation of any substitution variable
- **Profile variation in the outer loop**
 - Dummy parameter to define number of profile updates

Application – Hollow wear

- **14 simulations per step (164000 km, 364 steps)**

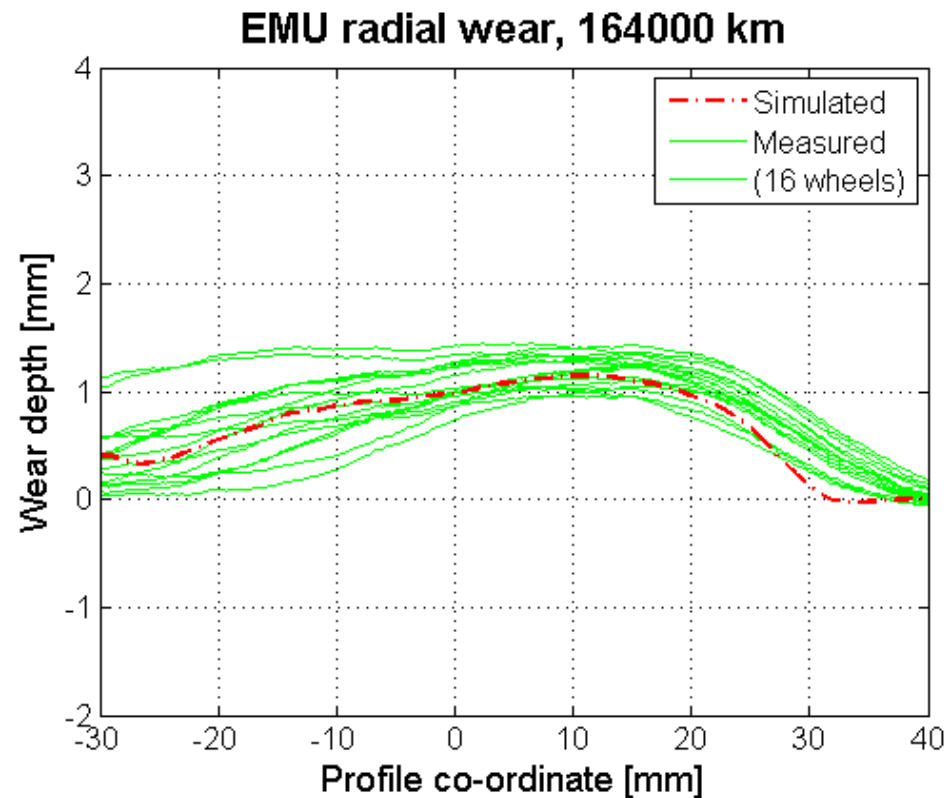
- Curves and tangent
- Tractive forces

- **Calibration of wear map**

- Ambient conditions
- Contact modelling

- **Corrective actions**

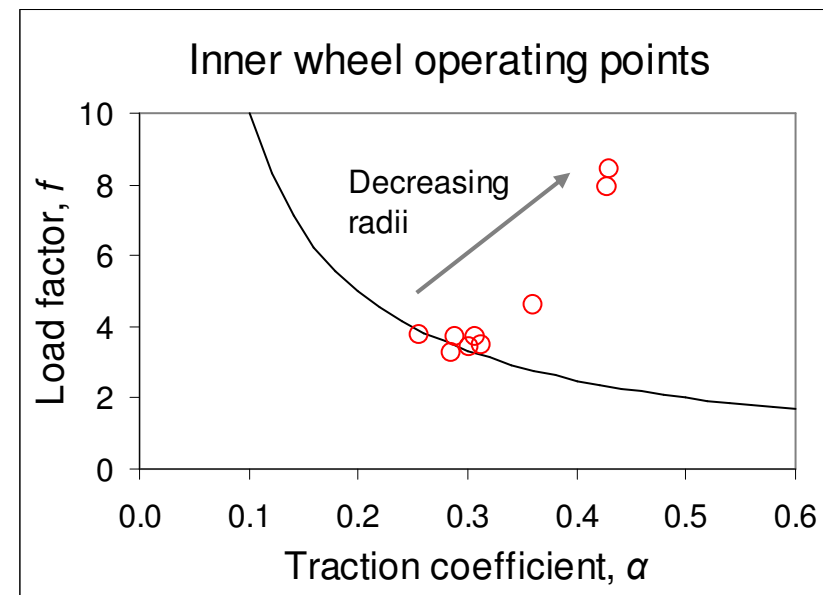
- Improved yaw dampers
- Monitoring of profile shapes



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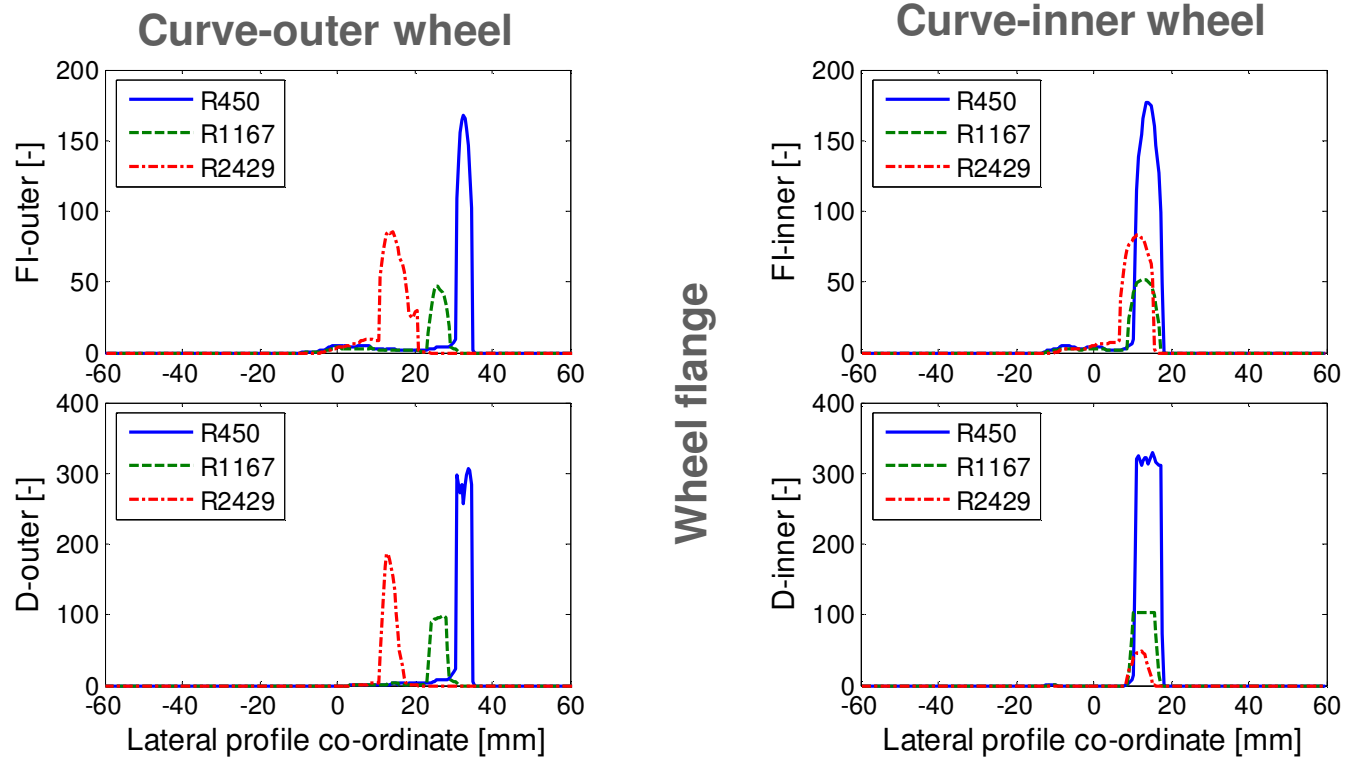
Application – Rolling contact fatigue

- **Three vehicles with different occurrence of RCF**
- **Evaluation of fatigue indices against reality**
 - Significance of wheel profile
 - Significance of wear
- **Quasi-static curving**
 - Curve radii 300 – 1000 m



Application – Fatigue accumulation

- High speed train during curve negotiation
 - Accumulated wear indices



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SIMPACK contact modelling

- **Classic contact modelling experience**

- One-point contact with quasi-elastic model works well
- Standard multi-point contact may cause problems due to the rigid flange contact approach (difficulties to find the correct contact location on wearing profiles)
- The all-elastic s-variable contact approach is more stable for multi-point cases
- The contact position and wear depth at the flange may be sensitive to the quasi-elastic regularisation setting (EPSREG)

- **New wheel / rail contact model being tested**

- Variable number and arbitrary location of contact points
- Expected to ease the difficulties experienced so far

- **Non-elliptic / conformal contact not available**

Conclusions

- **The effort to implement academic results in engineering should not be underestimated**
- **Experienced methods have been helpful in gaining understanding of the deterioration processes**
- **Wear models reasonably well calibrated for normal operating conditions**
- **RCF models indicate risk for appearance**
- **Improvements needed**
 - Contact model
 - Wear maps for extended range of contact conditions
 - Fatigue life model